



International Institute for Cavitation Research

## **Different Modelling Approaches for Flash Boiling**

**I.K. Karathanassis, P. Koukouvini, N. Kyriazis, M. Gavaises**

School of Mathematics, Computer Science and Engineering, City University London,  
Northampton Square, EC1V 0HB London, UK

# Presentation Layout

## ○ Problem Statement

- Effect of flash-boiling onset on Nozzle flows
- Technological Applications/Experimental investigations

## ○ Modelling approaches

- Thermodynamic equilibrium or not
- Suitable mass-transfer models

## ○ Benchmark cases

- Formulation of the numerical models
- Comparison to experimental data

## ○ Liquid Oxygen Properties

- Tabulated Data based on the Helmholtz EoS
- Preliminary Simulations

## ○ Concluding remarks

- Main findings
- Open questions



## Effect of flashing

- **Flash boiling:** Intense nucleation occurring throughout the entire liquid volume and usually caused by a rapid depressurization process that leads to liquid superheat, i.e.  $p=p_{\text{sat}}$  yet  $T>T_{\text{sat}}$ .
- **Jet atomization:** Rapid disintegration of the liquid bulk into small droplets; formation of a fine spray for flashing conditions
- **Occurrence of flashing in nozzle flows is associated to:**
  - Reduced penetration of the jet exiting the nozzle
  - Enhanced spray atomization efficiency
  - Increased spray angle
  - Complex pressure distribution at the spray region (formation of shockwave system)



# Exp. Investigations/Applications

## Operation under (partial) vacuum conditions

- Gasoline engines
- Rocket engines

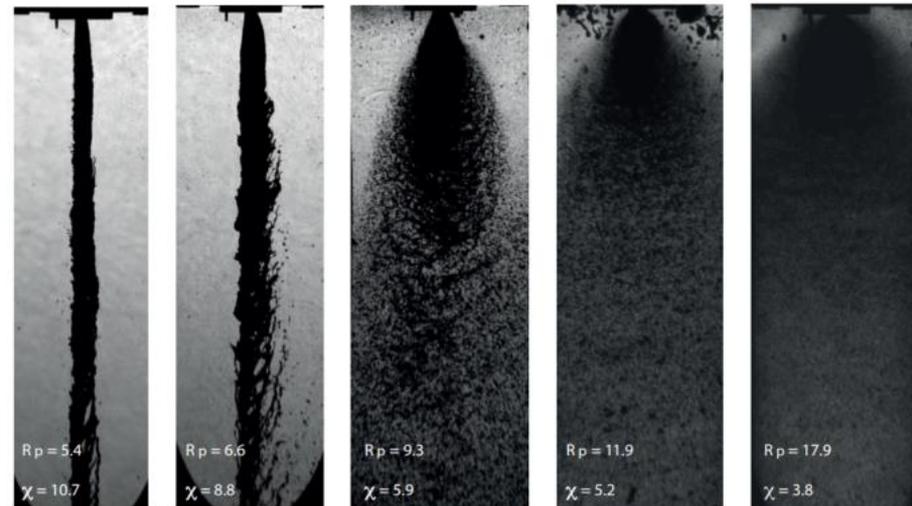
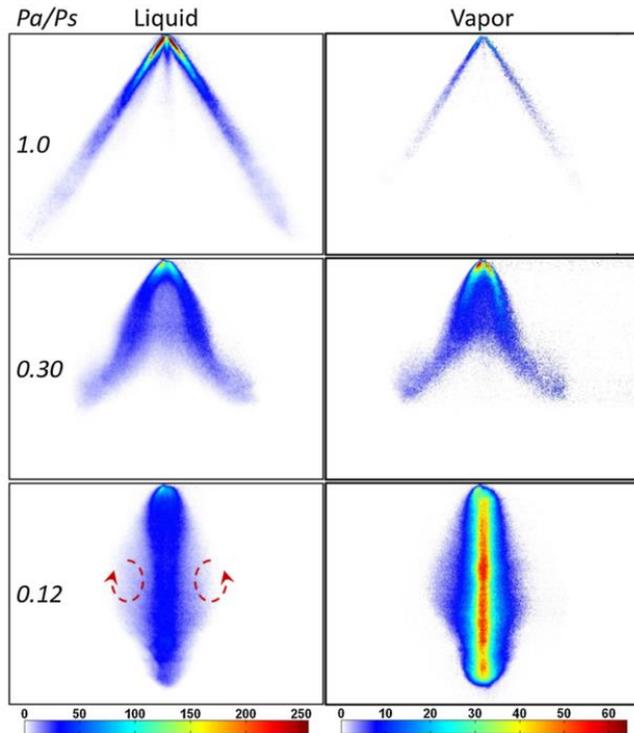


*Flashing is possible to occur in both applications yet the operation objectives are in fact opposite*

Zhang et al., 2014: n-hexane flashing at  $T_{inj}=298-358K$  (LIEF)

Common macroscopic flow features despite the difference in thermodynamic properties

Lamanna et al., 2015: LOX flashing at  $T_{inj}=94K$  (Shadowgraphy)





# Modelling Phase-Change

Two-phase models: Finite mass-transfer rates associated with empirical quantities

Linear mass-transfer rate (Knudsen)

$$\dot{R} = C_{evap} A_{int,tot} \alpha_l \rho_l \Delta p$$

$$A_{int,tot} = n_0 A_{bubble}$$

Hertz-Knudsen Equation

$$\dot{R} = \frac{\lambda(p_{sat} - p)}{\sqrt{2\pi R_g T_{int}}}$$

$\lambda < 1$ : departure from equilibrium

Homogen. Relaxation Model (HRM)

$$\dot{R} = -\rho_{mix} \frac{Y - Y_e}{\Theta}$$

$$\left\{ \begin{array}{l} Y_e = \frac{h_{mix} - h_{sat,l}}{h_{sat,v} - h_{sat,l}} \\ \Theta = \Theta_0 a^m \phi^n \end{array} \right.$$

Homogen. Equilibrium Model (HEM)

$$p(\rho, T) = \begin{cases} B \left[ \left( \rho / \rho_{sat,L}(T) \right)^n - 1 \right] + p_{sat}(T), & \rho \geq \rho_{sat,L}(T) \\ p_{sat}(T), & \rho_{sat,V}(T) < \rho < \rho_{sat,L}(T) \\ \rho RT, & \rho < \rho_{sat,V}(T) \end{cases}$$

# Numerical Formulation/ Bubble dynamics models

## 2D, transient flow simulations

- Mixture two-phase model: mechanical equilibrium between the phases
- Liquid: Tait equation of state
- Vapour: ideal gas

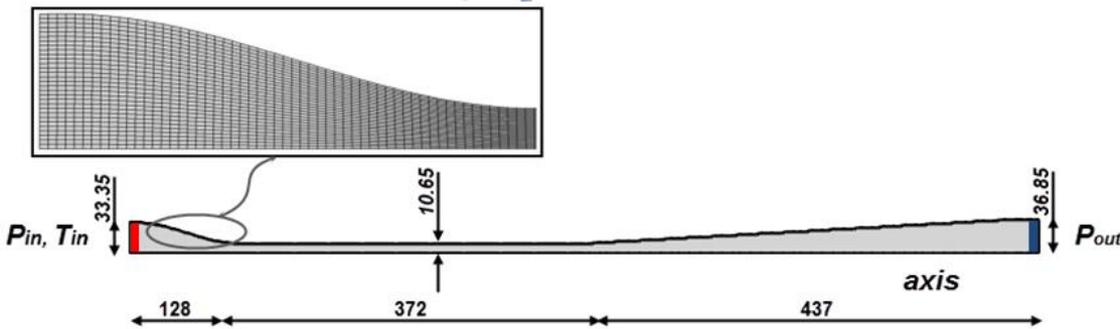
- Mixture compressibility: 
$$\frac{1}{\rho} \cong \frac{1-a}{\rho_l c_l^2} + \frac{a}{\rho_v c_v^2} - \frac{\delta m}{\delta p} \left( \frac{1}{\rho_v} - \frac{1}{\rho_l} \right)$$

Vapour fraction transport equation

$$\frac{\partial(a\rho_v)}{\partial t} + \nabla(a\rho_v \mathbf{u}) = \dot{R} \longrightarrow \text{Mass-transfer term modelled}$$

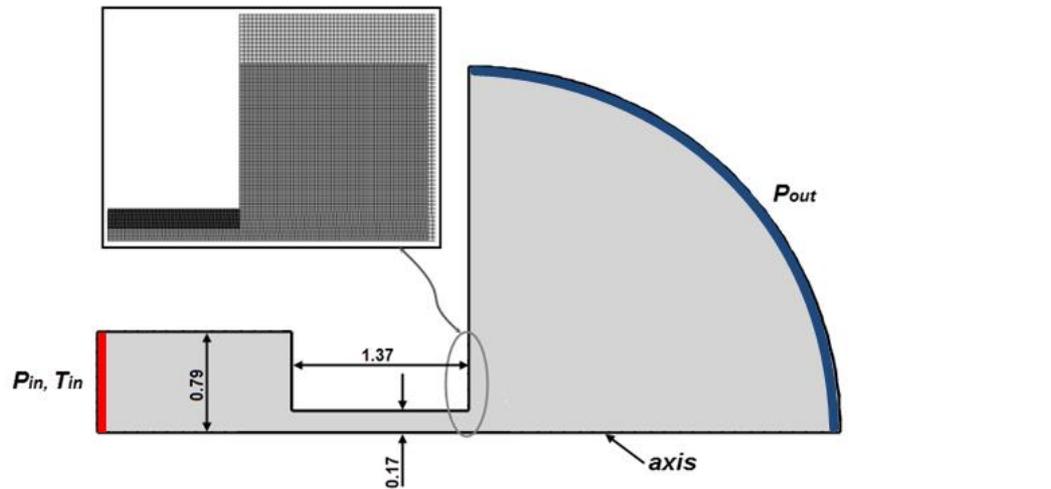
- SST k- $\omega$  turbulence model
- Implicit, coupled solver
- Second order schemes for momentum and turbulence advection
- Time step of  $1\mu\text{s}$  corresponding to CFL numbers  $\sim 4-5$  in the nozzle

# Benchmark Geometrical Layouts



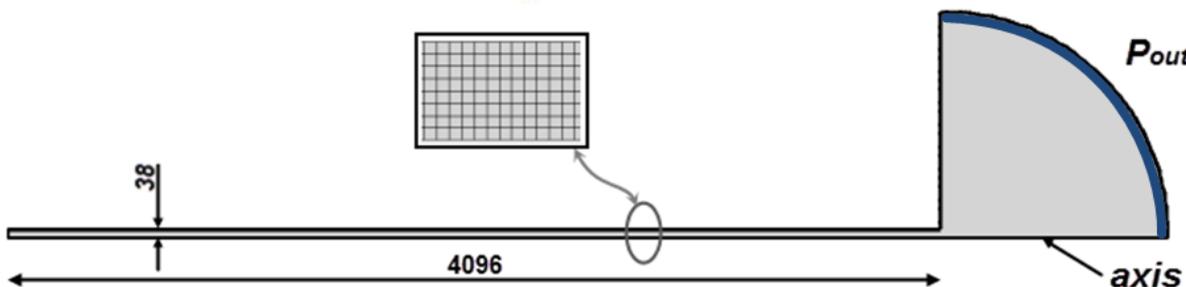
## Moby Dick nozzle

- Water, steady problem
- $T_{in} = T_{sat} - 2K$
- $P_{in} = 20\text{bar}, P_{out} = 5\text{bar}$



## Reitz nozzle:

- Water, steady problem
- $T_{in} = 360-427$
- $P_{in} = 7.97\text{bar}, P_{out} = 1\text{bar}$



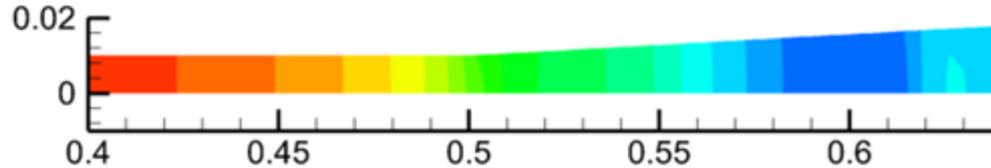
## Edwards' pipe:

- Water, transient problem
- $T_{in} = 502K$
- $P_{in} - P_{out} = 70\text{bar}$

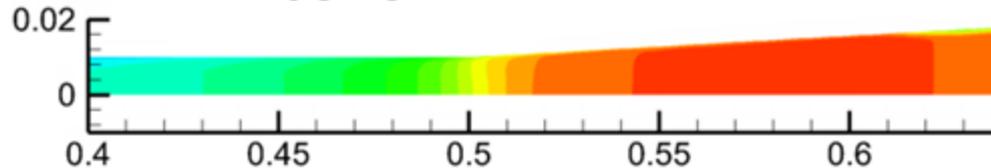


Hertz-Knudsen model,  $\lambda=0.1$

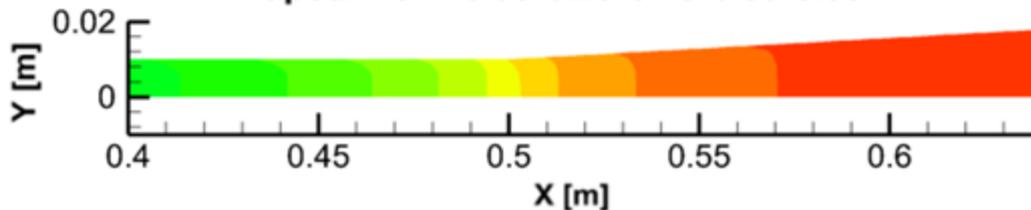
Pressure [Pa]: 300000 800000 1.7E+06



u velocity [m/s]: 10 25 40 55 70 85 120



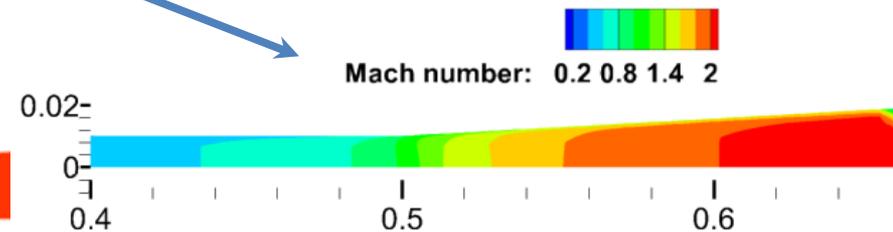
vapour vof: 0.05 0.25 0.45 0.65 0.85



## Moby-Dick Nozzle (1)

- Flow acceleration at the divergent region  $\rightarrow$  expansion of initially choked two-phase flow
- Supersonic flow downstream the throat
- Almost full liquid vaporization has occurred at an axial distance  $X=0.65\text{m}$

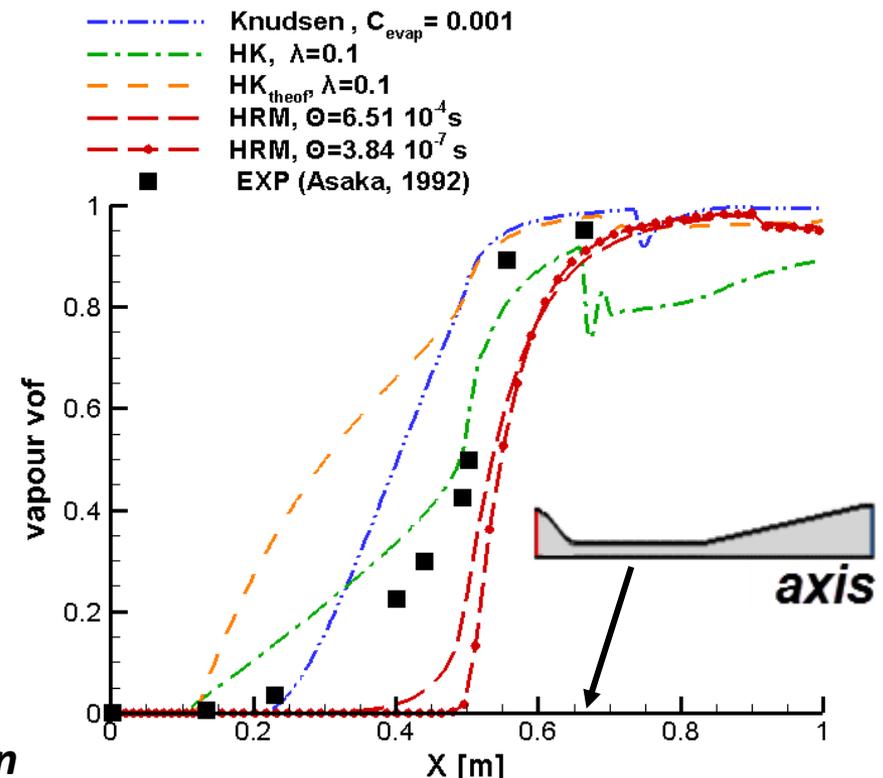
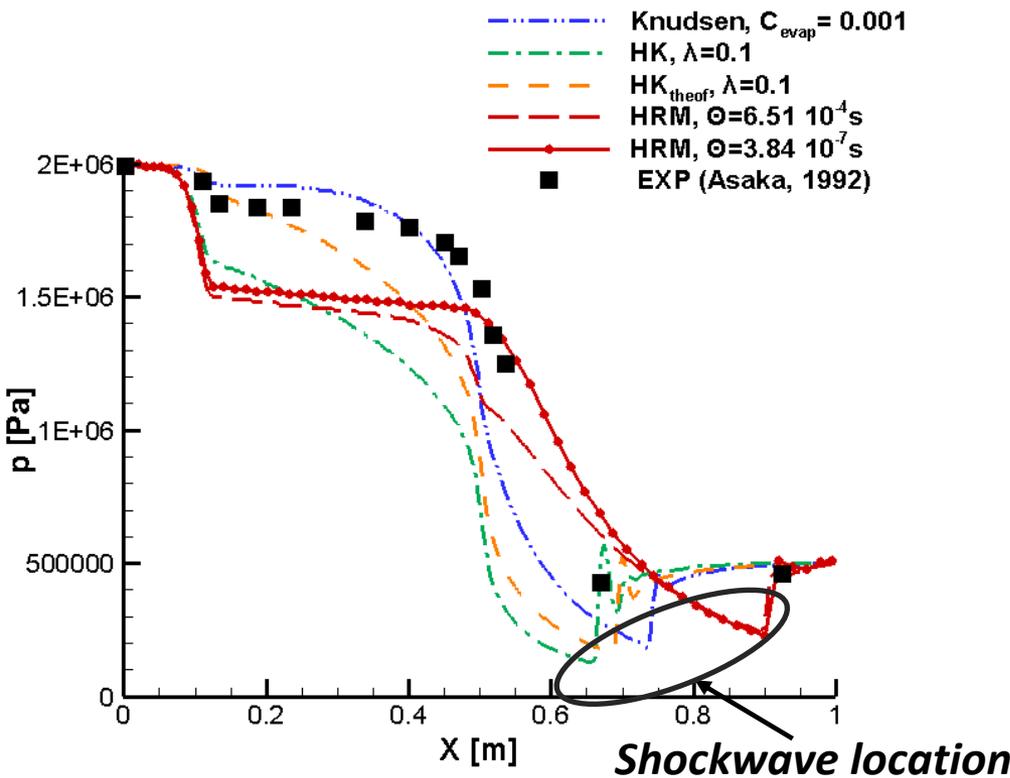
### Mach number distribution



# Moby-Dick Nozzle (2)

## Pressure/VOF distribution at the nozzle axis

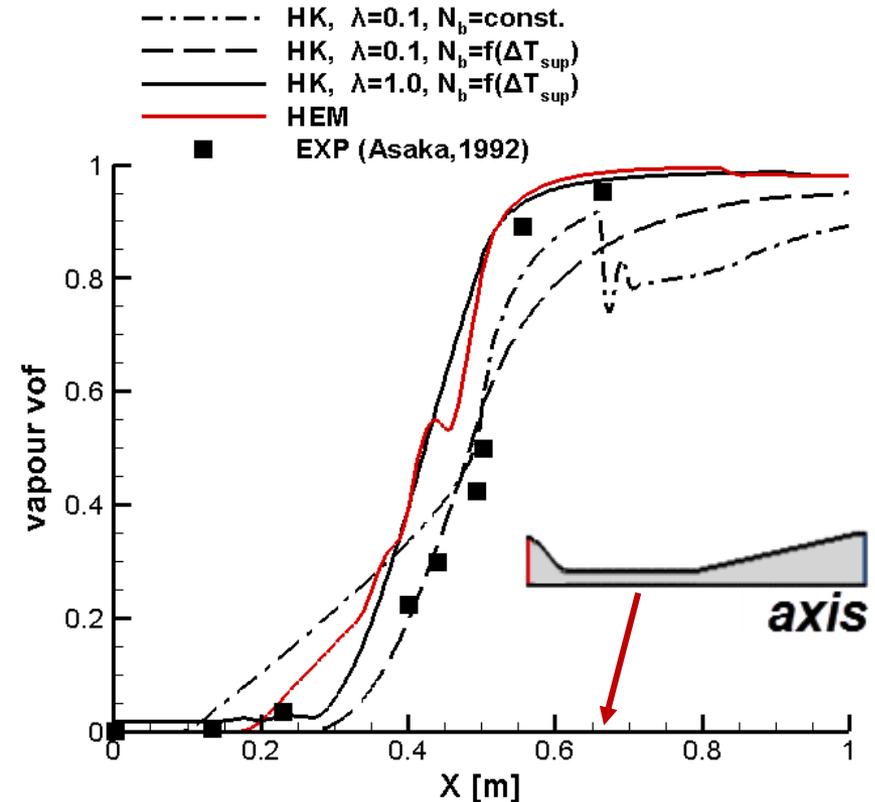
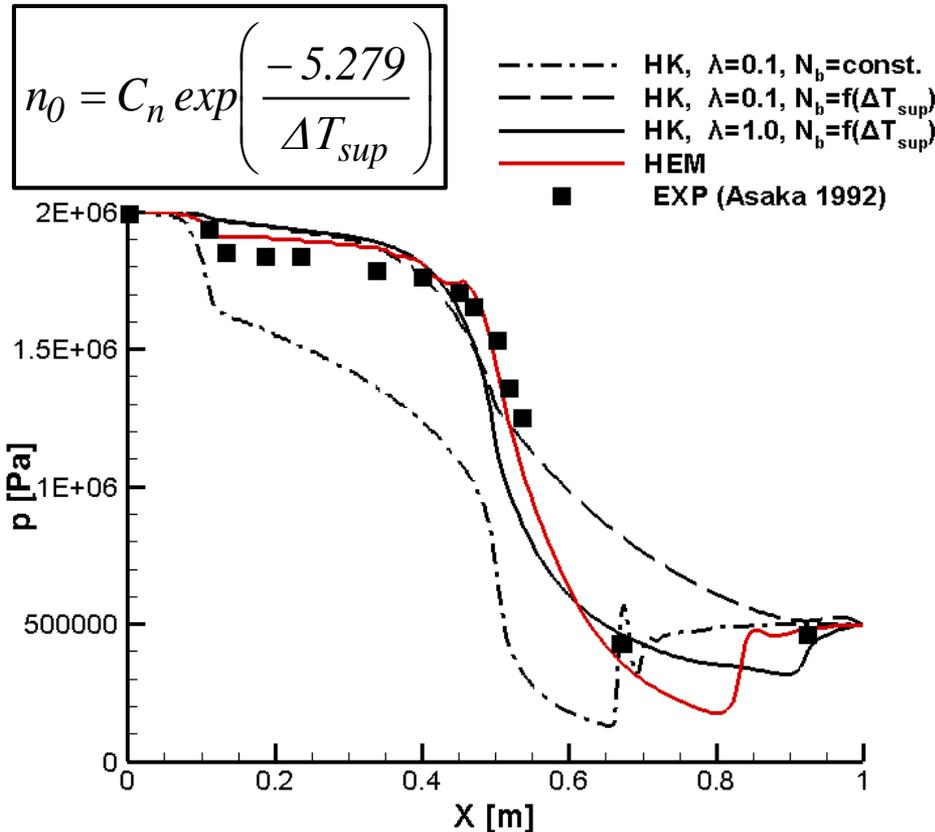
- Adequate agreement achieved by all models.
- Higher mass transfer rate accompanied by increased in-nozzle pressure
- Shockwave predicted by all models however at different locations



# Effect of nucleation-site density

## Variable nucleation site density (Senda et al., 1994)

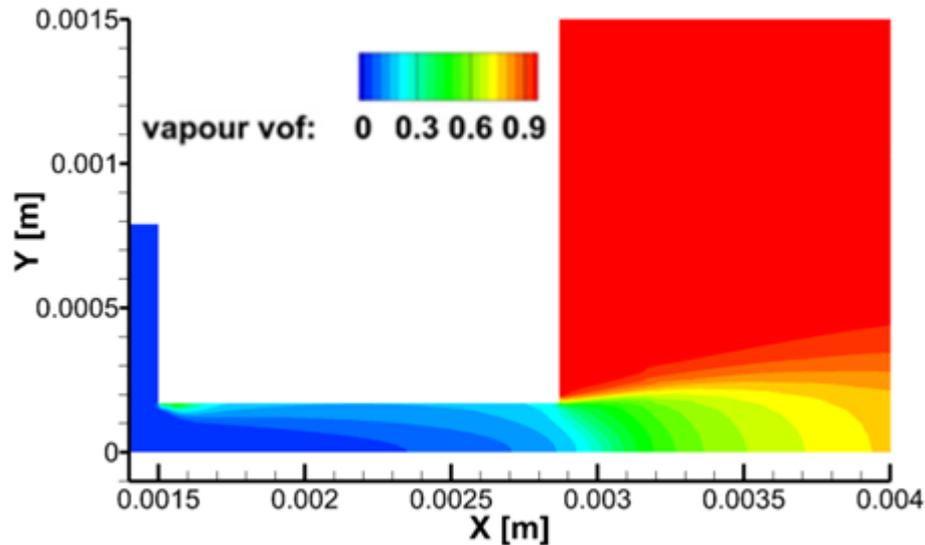
- Excellent agreement obtained for  $\lambda=1$  (thermodynamic-equilibrium)
- No clear distinction of the effects of nucleation-site density and actual thermodynamic conditions on phase-change rate



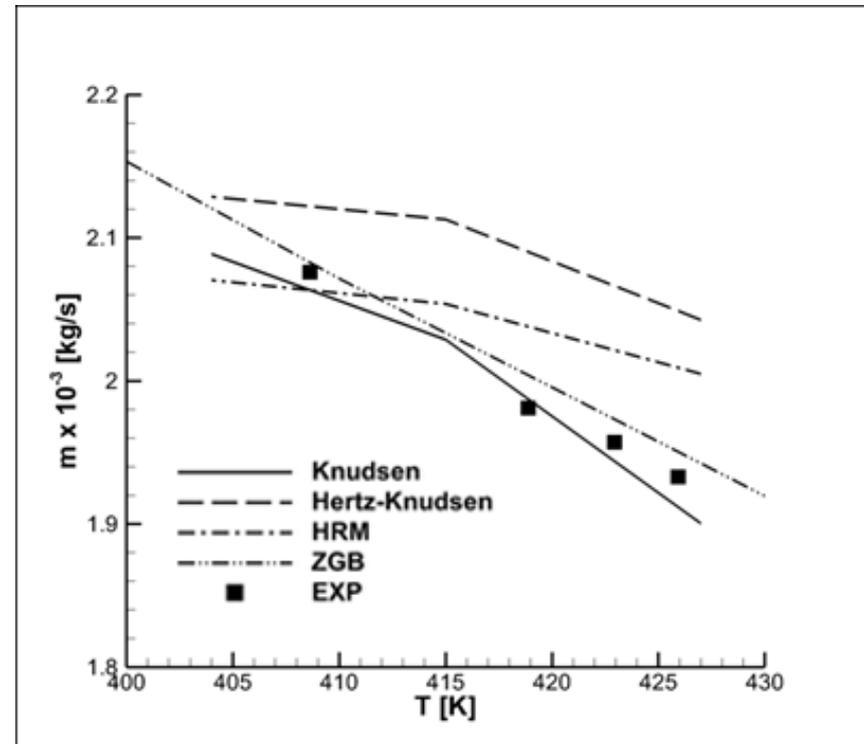
# Reitz nozzle(1)

## Phase field for T=427K

- Liquid core could evident at the nozzle outlet and severe atomization sets in immediately downstream the nozzle outlet
- Mass flow rate decreases due to the increasing part of nozzle cross section occupied by vapour



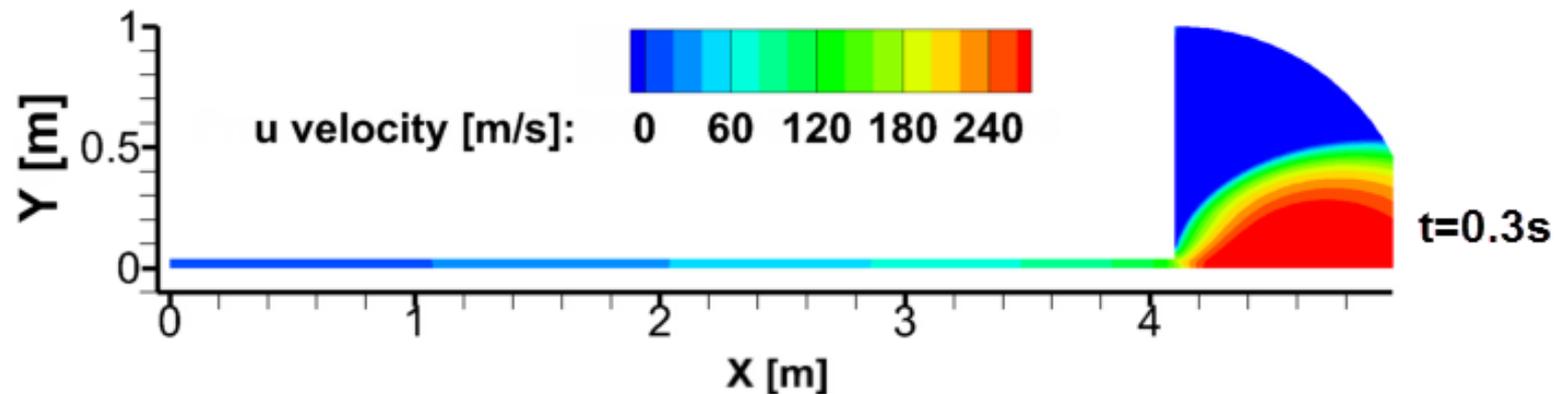
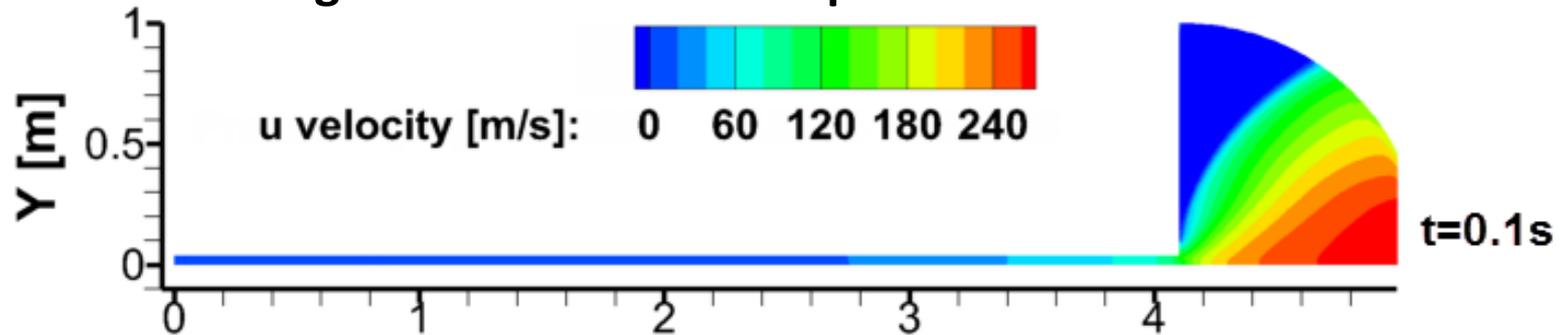
T=427K, HRM ( $\Theta=6.51 \cdot 10^{-4}$ s)



# Edwards' pipe (1)

## Velocity field (Hertz-Knudsen, $\lambda=0.1$ )

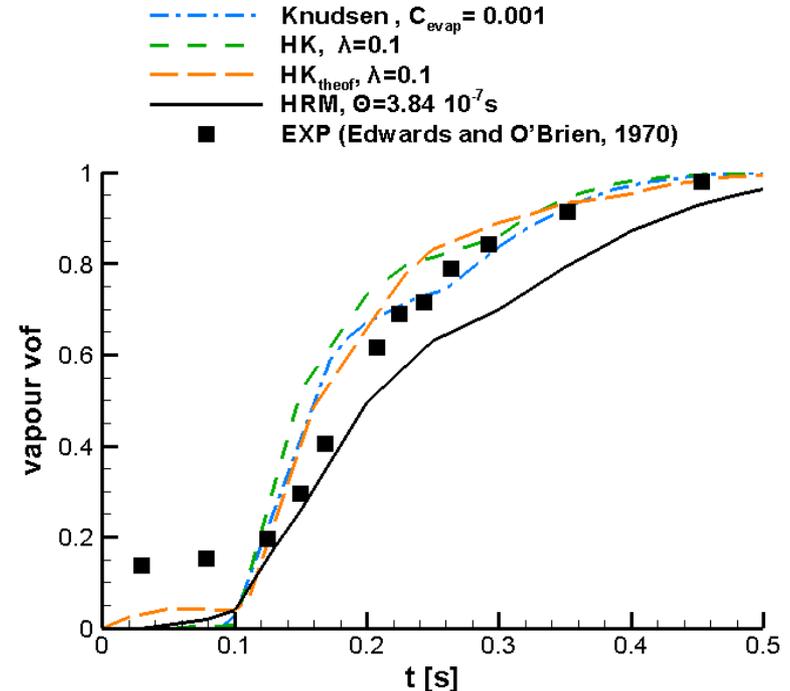
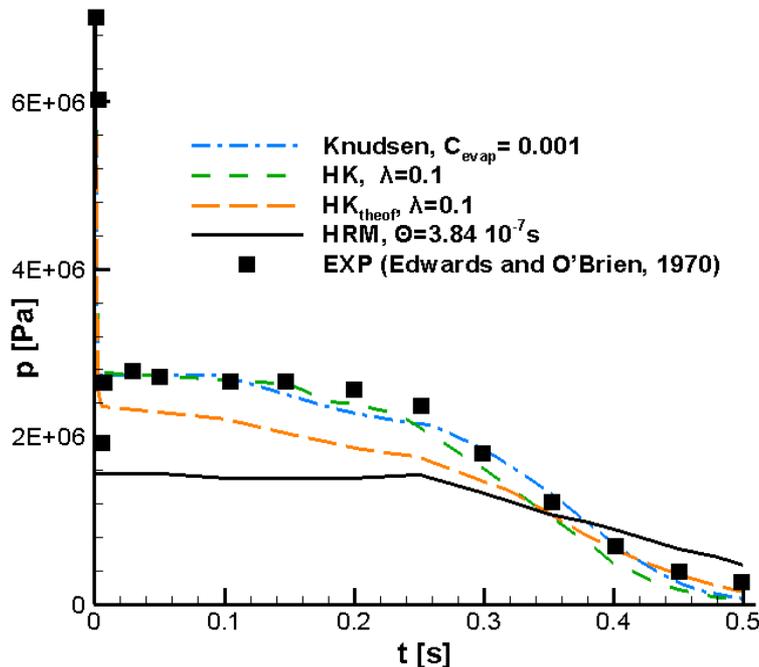
- Flow acceleration towards the pipe outlet as the phenomenon evolves
- Increased cone angle due to flow over-expansion



# Edwards' pipe (2)

## Pressure at the pipe left wall/Vapour vof at the pipe mid-section

- Knudsen and Hertz-Knudsen equations produce accurate predictions
- Liquid vaporization caused due to rarefaction wave
- Vaporization occurs at a sharp interface reaching left wall at  $\sim 0.5$  s
- Calibration of HRM parameters for externally flashing flows needed



## Conclusions

- **Accurate predictions of Hertz-Knudsen models for all examined cases. Possible HRM tuning for mass transfer through sharp interphase**
- **Information of nucleation-site density is vital for accurate predictions.**
- **Flashing within the nozzle is accompanied with choked flow and increased spray cone angle of the expanding jet.**
- **LOX thermodynamic properties successfully predicted using Helmholtz EoS; Good agreement with NIST REFPROP**
- **Preliminary simulations verified the robustness of the implemented methodology for predicting flashing of cryogenic fuel**

## Open Questions/ Future research

- Clear identification of the flashing regimes, i.e. internal or external and effect of orifice manufacturing characteristics (L/D, roughness etc.)
- Characterization of the inception-point distribution in the liquid bulk- Methods to avoid model tuning
- Effect of heat-transfer rate & Jacob number on mass-transfer rate
- Explicit correlation between the expanding-jet characteristics (velocity, cone angle, droplet size) and the phase-change rate within the nozzle
- Experimental studies needed to designate HRM coefficients for hydrocarbon and cryogenic fuel